

某汽轮机轴封汽外泄的原因分析及处理

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摘要:通过对某汽轮机轴封泄漏征象及大修中发现情况的分析, 得出该机轴封泄漏的根本原因, 并提出了一种经济、简易、有效的处理方法。

关键词:汽轮机; 轴封; 泄漏量; 处理

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1 泄漏情况

某热电厂 3 号汽轮机 (C6-3.43/0.49) 系调整抽汽机组, 于 1999 年 11 月投入运行。运行过程中发现, 该机前轴封处有大量蒸汽外泄, 冷态启动时车头可以看到大量白汽冒出, 正常运行时, 前轴承靠近汽缸侧 1/3 处总呈湿润状态, 同时, 透平油回油观察窗上有水珠凝结。运行一个月后, 化验发现透平油已大量带水, 严重乳化, 油箱底部放水阀中放出积水约 500 kg (经化验为凝结水), 尽管进行定期的滤油、脱水, 仍不能解决问题, 到 2000 年 3 月大修时, 已多次发生调速系统卡涩事故。大修中发现:

(1) 推力瓦已磨损掉 0.7 mm (汽机轴向位移早在年初已处报警状态);

(2) 油管、油箱调速系统严重结垢, 油箱底部油泥厚达 5~10 mm;

(3) 前轴封处汽缸洼窝中心偏差达 0.7 mm, 此处汽封最大间隙为 2 mm, 平均 1.5 mm, 后汽封因安装质量不良, 呈倒伏状。

大修时针对轴封汽外漏的缺陷作如下处理:

将洼窝中心偏差调整至 0.05 mm 以内; 更换全部汽封片并调整间隙至 0.25~0.30 mm (要求 0.30~0.35 mm, 为减少泄漏量, 在征得生产厂家同意后减少 0.05 mm); 自发电机冷却风扇引出两根直径为 15 mm 的导管, 分别接入前后轴承挡油环内。

大修后启动发现:

开机时车头仍有白汽冒出, 但较原来明显减少,

前轴承箱近汽缸端微湿; 回油观察窗中仍有水珠凝结; 挡油环中引入鼓风后, 轴承箱排气口油烟有所增加; 运行两周后, 化验发现透平油中含水量为 3% 左右, 并从油箱中放出大量积水。

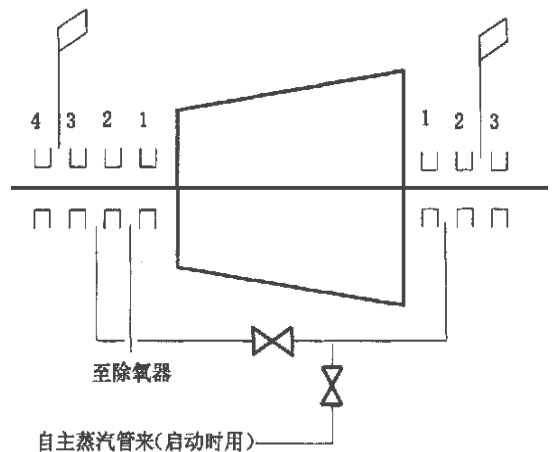


图 1 汽轮机轴封系统

2 原因分析

(1) 该机轴封系统如图 1 所示。为保证后轴封处于微正压状态, 前轴封第二、三挡之间设计压力为 0.002~0.003 MPa (表压, 下同), 而在大修前, 须达 0.025 MPa 以上时才能保证后轴封信号管微微冒汽; 大修调整轴封间隙后, 仍需保证前轴封 0.01 MPa 以上的压力, 才能维持后轴封的微正压。根据文献 [1] 中的轴封泄漏量计算公式, 求得设计工况下后汽封向排汽缸漏汽量约为 40 kg/h (大修前泄漏量更大), 而后轴封进汽口最小处直径仅为 15 mm, 以蒸汽温度 120 °C 记, 相应蒸汽流速为 128 m/s, 远高于同类机组的 6.01 m/s, 也高于管道设计规范中 45~60 m/s 的要求。

(2) 对前汽封而言, 第二、三挡轴封处压力为

0.01MPa 时, 泄漏量为 24 kg/h, 但前汽封向空排汽口最小通径为 20 mm, 以蒸汽温度 320 °C 记, 相应流速为 70 m/s, 同样高于同类机组 6.01 m/s 的设计流速。

(3) 前轴承箱距汽缸仅 100 mm, 且有汽机座架, 保温等阻拦, 汽机运行时轴的鼓风作用又使挡油环与轴的间隙处存在一定的负压, 对蒸汽有抽吸作用。

综上所述, 造成轴封汽外泄的原因在于汽机后轴封供汽通道过小, 使前轴封压力超过设计值, 而前轴封向空排汽的局部通道过细, 又使前轴封排汽信号管产生的通风力不足以排出全部蒸汽, 从而有一部分蒸汽自前轴封第四挡外泄; 轴承箱内的负压又将泄出的蒸汽吸入其中, 引起透平油中带水。

前后轴承箱挡油环处通入发电机风扇的鼓风, 其目的在于迫使此处产生一定的正压, 阻止蒸汽进入, 但因挡油环间隙较大(约为 1 mm), 鼓风从其上部引入后, 不能保证在整个圆周上维持正压, 因而不能有效地阻挡蒸汽。

3 解决办法(图 2)

(1) 前轴承箱隔热板后加一挡汽板 1, 与轴保留 0~0.1 mm 的间隙;

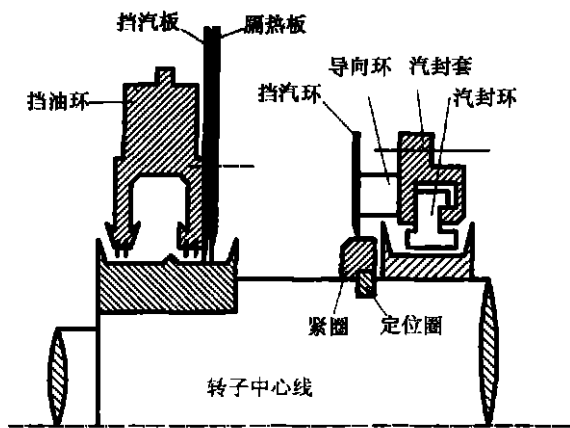


图 2

(2) 做一导向环, 外径同汽封套相同, 向上开一大于轴径 5 mm 的缺口, 一面固定于汽封套上(最好在大修时用螺丝固定, 应急处理时可在结合面涂上耐热涂料后与汽封套上半点焊固定);

(3) 导向环另一面另装一挡汽环(用螺钉固定), 与轴之间隙为 0~0.1 mm。

为防止挡汽板与轴之间发生磨擦而损伤转子, 挡汽板用工业纯铝(L05)制成。这样, 挡汽板减少了因抽吸而进入轴承箱的气体总量, 而导向环与挡汽环在汽机转子外伸端形成了一向上开口的腔室, 汽机一旦运行, 转子表面与空气间的磨擦作用使轴面附近的气体随轴产生圆周运动, 并在离心力的作用下由腔室上开口处逸出, 使腔室内的靠近轴表面处呈负压状态, 可靠地阻止蒸汽由挡汽板与轴之间的间隙处向轴承箱泄漏。导向环开口处的流通面积是汽封间隙截面的 20 倍, 因此即便汽封间隙由 0.3 mm 增大至 1~1.5 mm, 导向环与挡汽板仍能有效地防止蒸汽外泄。

另外, 汽机正常运行时, 前轴封处的温度肯定超过 100 °C, 故不用考虑蒸汽凝结的问题, 但若此方法用于后轴封, 则必须在导向环上开适当的疏水孔并将凝结水导出。

汽机进行上述改造后投入运行, 运行中对透平油进行过滤、脱水处理。启动时发现, 从前轴封泄出的蒸汽及抽吸进入的少量空气沿右上 45° 方向逸出(自进汽端向发电机端看, 汽机转子转向为顺时针), 前轴封箱干燥无水。运行三个月后, 进行透平油油质化验, 油中带水现象消失。

4 结语

汽轮机轴封泄漏现象在各类机组中经常可以碰到, 甚至于一些具有较为完善轴封系统的大型机组如 300 MW 机也不能例外^[2]。因轴封与轴承箱之间间隙一般较小, 轴封泄漏时往往有蒸汽进入透平油中, 引起更为严重的事故。若泄漏由设计原因引起且在本体上改造较为困难, 或泄漏需大修时才能消除, 但一时又无法大修的, 上述方法不失为一种经济、简易、有效的手段。另外, 生产厂家在设计轴封系统时, 应考虑各种阻力。

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(辉 编辑)

Circulating Fluidized Bed Boiler Furnace by Utilizing a Monte Carlo Method [刊, 汉] / SUN Yong-li, HE Yu-rong, LU Hui-lin (Thermal Energy Engineering Department, Harbin Institute of Technology, Harbin, China, Post Code: 150001), TAN Xiu (Electric Power Scientific Research Institute of Jilin Province, Changchun, China, Post Code: 130021) // Journal of Engineering for Thermal Energy & Power. —2001, 16(6)—650~652

A numerical study was conducted of the heat transfer in a circulating fluidized bed boiler furnace. The relevant model being set up takes into account the influence of the concentration distribution of axial and radial particles. The calculation of the model reveals the distribution variation of flue gas concentration and heat flux density within the furnace. The results of the calculation indicate that in the heat transfer calculation of a circulating fluidized bed boiler furnace the convection heat transfer of particle phase should not be neglected. **Key words:** circulating fluidized bed boiler, heat transfer, Monte Carlo method

燃煤工业锅炉湿式旋流烟气脱硫装置的数值模拟法优化设计 = Optimized Design of a Numerical Simulation Method for the Vortex Desulfurization Device of Wet Flue Gases of a Coal-fired Industrial Boiler [刊, 汉] / QIU Zhong-zhu, XU Ji-huan, ZHANG He-sheng (Thermal Energy Engineering Department, Tongji University, Shanghai, China, Post Code: 200092) // Journal of Engineering for Thermal Energy & Power. —2001, 16(6)—653~655, 676

Through the use of a $k-\epsilon$ dual equation model a numerical simulation was conducted of the vortex area speed and pressure field of a vortex wet gas desulfurization device. Meanwhile, with the help of a single-particle dynamic model the movement of liquid drops in the gas flow field was simulated, and by utilizing the numerical simulation method the structural parameters of the above-mentioned vortex device were determined. As a result, the optimized design of the vortex wet gas desulfurization was realized, contributing to a reduction of test expenses and a decrease in experimental work load. **Key words:** desulfurization device, vortex area, numerical simulation, structural parameter

船用蒸汽动力装置控制监测系统的研制 = The Development of a Control and Monitoring System for a Naval Steam Power Plant [刊, 汉] / LI Lai-chun, LIU Fan-ming, et al (Harbin No. 703 Research Institute, Harbin, China, Post Code: 150036) // Journal of Engineering for Thermal Energy & Power. —2001, 16(6)—656~658, 674

Due to its complicated system, the presence of many equipment items and the large quantity of needed control parameters a steam power plant has a majority of its controlled objects not liable to be represented by simple mathematical models. During the operation of the plant many parameters are interrelated and involved in a complicated coupled relationship. Numerous controlled parameters cannot meet usage requirements if a single loop control is employed. Moreover, a naval steam propulsion plant features a high frequency of load changes and a wide range of such changes, resulting in a control system, the implementation of which demands sophisticated technical skills. In light of the above the authors present the composition and functions of a centralized control and monitoring system for a naval steam propulsion plant along with some innovative approaches of that system. **Key words:** naval vessel, steam power plant, automatic control

某汽轮机轴封汽外泄的原因分析及处理 = An Analysis of the Cause of a Steam Turbine Shaft Seal Leakage and Its Treatment [刊, 汉] / Zhang Shao-bo (Cixi Thermal Power Plant, Cixi, Zhejiang Province, China, Post Code: 315300) // Journal of Engineering for Thermal Energy & Power. —2001, 16(6)—659~660

After an analysis of the symptoms of shaft seal leakage and other abnormal conditions detected during the overhaul of a steam turbine the root cause of the leakage was identified and an economic, simplified and effective method proposed for its resolution. **Key words:** steam turbine, shaft seal, leakage rate, problem solving

轴向型粗粉分离器改进与完善 = Improvement and Advancement of an Axial Type Separator of Coarse Pulverized Coal [刊, 汉] / LU Tai, CHOU Lin-qing, CHEN Fu, et al (Jilin Electric Power Institute, Jilin, China, Post Code: 131200), NIU Zhi-hong (Jilin Municipal Thermal Power General Co., Jilin, China, Post Code: 131200) // Journal of Engineering for Thermal Energy & Power. —2001, 16(6)—661~663