

扇形孔出口宽度对气膜冷却效率影响

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摘 要: 数值模拟了不同出口宽度的扇形孔射流在不同吹风比下的气膜冷却效率, 侧重于分析涡流结构和涡量对冷却效率的影响。结果表明, 出口宽度增大不但减小了涡量, 还改变了涡流方向。出口宽度增大到 2.5 倍入口直径时, 孔中心下游冷气流线由燃气指向壁面, 有效改进冷气贴附性。气膜孔出口宽度越大, 冷却效率越大。随着吹风比的增加, 气膜孔出口宽度对冷却效率影响越明显。

关 键 词: 涡轮叶片; 气膜冷却; 涡量; 冷却效率; 数值模拟

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引 言

燃气轮机涡轮叶片冷却的主要技术之一是气膜冷却。圆柱孔因其加工方便, 在早期涡轮叶片气膜冷却中得到了广泛应用。但是, 圆柱孔射流产生的对旋涡使冷气贴壁性较差冷却效率较低。通过改进气膜孔几何结构可以有效提高冷却效率。主要方法有两种: 一种是 20 世纪 90 年代朱惠人等研究了圆锥孔、簸箕孔下游气膜冷却特性, 当吹风比大于 0.7 时, 两种孔射流的冷却效率均优于圆柱形孔射流冷却效率^[1]。此后, 有关各种扩张孔射流的冷却效率报道非常多, 如姐妹孔减少孔口处回流而增加冷却效率。Console 型孔改变孔外涡流结构而提高冷却效率^[2~3]。这些孔形在提高冷却效率的同时也增加了加工工艺难度。另一种提高冷却效率的途径是在圆柱孔上增加其它结构^[4~7], 如气膜孔出口增加横槽, 三角突起, 这种途径在提高冷却效率同时并没有给加工工艺带来明显难度。最近提出的反涡孔改变了涡流方向而提高了冷却效率^[8]。

扇形孔射流的冷却效率好于圆柱孔射流冷却效率, 目前的工艺水平也完全可以加工出来。扇形孔射流由于增加了气膜孔出口面积而减小冷气射流流量, 从而减弱冷气和燃气掺混程度。以往研究的扇

形孔出口宽度相对较小, 射流后涡流方向和圆柱孔射流后涡流方向基本相同。本研究通过对扇形孔射流气膜冷却进一步研究发现, 孔出口宽度达到一定程度时, 孔外涡流强度不仅减弱, 流动方向还发生了变化。在此, 给出了宽口扇形孔气膜冷却效率结果以及详细的流场和温度场, 为扇形孔结构设计提高参考。

1 数值模拟的模型及条件

1.1 数值模拟

利用 Fluent 软件隐式求解法对 $N-S$ 方程进行求解, 采用 realizable $k-\epsilon$ 湍流模型和增强壁面函数。湍流模型的选取对计算结果有直接影响, 在气膜冷却数值模拟计算中 realizable $k-\epsilon$ 湍流模型已被很多学者认同。本研究将利用 realizable $k-\epsilon$ 湍流模型计算的结果和试验数据进行了对比, 吻合的非常好。压力和速度耦合采用 SIMPLE 算法, 各物理量的离散项均采用二阶迎风格式, 解收敛的标准是所有参数残差均小于 10^{-6} 。

1.2 计算域和网格结构

图 1 给出了气膜孔结构示意图, 射流角度为 45° , 扇形孔出口扩张角为 15° , 宽径比 $W/d = 2.5$, 厚径比 $T/d = 3$, 长径比 $L/d = 4.2$ 。图 2 给出了计算域, 包括主流通道, 气膜孔和供气腔, 主流通道只有一个径向周期, 宽度为 3 倍气膜孔直径, 即孔间距为 3。通道入口到主孔中心的距离为 10 倍孔径, 主孔中心到通道出口的距离为 40 倍孔径, 主流通道在 y 向距离为 10 倍孔径。供气腔在 y 向距离为 6 倍孔径, 在 x 方向为 10 倍孔径。由于本研究主要是计算壁面附近的传热特性, 壁面附近的网格质量直接影响计算结果的好坏。网格总数为 180 万, 第一层网格的 y^+ 在 1 - 5 之间, 使网格尺度满足增强壁面函

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数的要求。并且验证数值模拟的可靠性,为了对比所计算的扇形孔射流冷却效率和圆柱孔射流冷却效率的差别,将圆柱孔射流的计算结果和试验数据的对比在数据分析中给出。

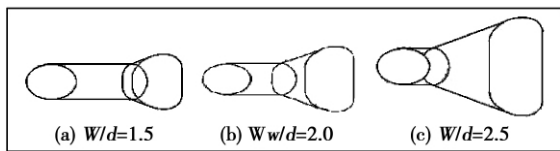


图 1 气膜孔结构

Fig. 1 Air film hole structure

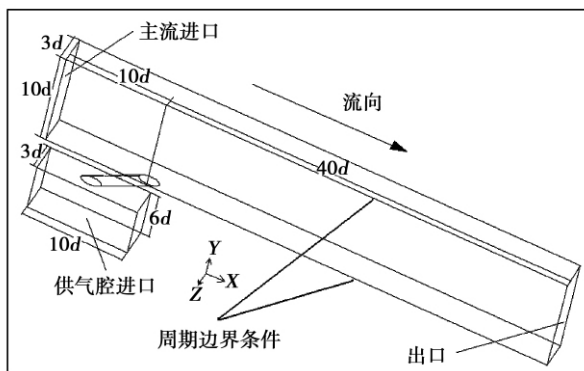


图 2 计算域

Fig. 2 Computation domain

1.3 参数定义和边界条件

气膜冷却效率的定义:

$$\eta_t = \frac{T_g - T_{aw}}{T_g - T_c} \quad (1)$$

式中: T_g —主燃气流温度; T_{aw} —气膜冷却下绝热壁温; T_c —冷气温度。

吹风比定义为:

$$Br = \frac{\rho_c u_c}{\rho_g u_g} \quad (2)$$

式中: ρ_c —冷气密度; u_c —气膜孔圆柱段的冷气平均速度; ρ_g —主流燃气密度; u_g —主流的入口速度。

主流入口定义为速度入口,给定平均速度为 15 m/s,温度为 330 K,出口为压力出口。冷气质量流量根据吹风比给出,其温度为 300 K,湍流度为 1%,吹风比分别为 0.5, 1.0, 1.5。

2 结果和讨论

2.1 径向平均冷却效率

图 3 给出了扇形孔出口宽度对径向平均冷却效

率的影响。坐标原点位于气膜孔出口中心,横坐标 x/d 为气膜孔下游到坐标原点的距离与扇形孔圆柱段直径的比值,纵坐标为径向平均冷却效率,即在相同的 x/d 位置,不同网格上的冷却效率的平均值 η_{ave} 。为了验证计算结果的可靠性,图 3 也给出了圆柱孔射流冷却效率计算值和实验数据的对比,计算结果和实验数据比较吻合,差别不到 10%。这说明所采用的湍流模型和网格数量在计算气膜冷却时可以得到较好的结果。本研究的扇形孔射流冷却效率的准确与否主要由孔下游的流动和传热特性决定,当圆柱孔射流的冷却特性比较准确时,扇形孔射流的冷却特性也应该是准确的。

从图 3 可以看出,出口宽度增加使冷却效率明显提高。随着吹风比的增大,出口宽度对冷却效率影响增大。以 $x/d = 10$ 为例,宽径比 W/d 从 1.5 增加到 2.5,吹风比 0.5 时冷却效率增加了 0.13(图 3 (a)),吹风比 1.0 时的冷却效率增加了 0.25(图 3 (b)),吹风比 1.5 时的冷却效率增加了 0.35(图 3 (c)),这说明吹风比越大,出口宽度增加使冷气利用率提高越多。

不同出口宽度的冷却效率沿着流动方向都逐渐降低,吹风比越小,冷却效率降低越快。出口宽度对冷却效率影响随着 x/d 增大而减小,这是由于冷气在壁面贴附较好,随着冷气向下游运动,冷气层逐渐被燃气稀释而导致冷却效率降低,吹风比越小,冷气层被燃气稀释越快。

需要指出的是,所计算的冷却面是平面,这与真实情况下涡轮叶片的曲面形状是有差别的,但是由于其主要研究的是气膜孔出口宽度对冷却效率的影响,结果从定性角度反映出来的规律与真实情况下应该是接近的。

图 4 给出了吹风比 1.0 时,气膜孔下游 $x/d = 5$ 位置不同出口宽度射流下的速度矢量云图,云图为当地的全速度与通道入口速度的比值。从速度云图可以看出,随着气膜孔出口宽度增加,对漩涡逐渐减弱,较低的无量纲速度范围逐渐向着孔两侧扩散。 W/d 为 1.5 和 2.0 时,存在明显的对漩涡结构,将高温燃气翻卷到冷气底层; W/d 为 2.5 时,形成了两对对漩涡结构,在气膜孔出口的中心,速度矢量指向壁面,将燃气推向孔两侧,减少了与冷气掺混的燃气体量,有效地改进冷气在壁面的覆盖,孔出口两侧存在的翻卷涡尺度非常小,对冷气在壁面覆盖的负面影响较小。这也是本研究的扇形孔射流冷却效率不同于以

往扇形孔射流冷却效率机理所在。

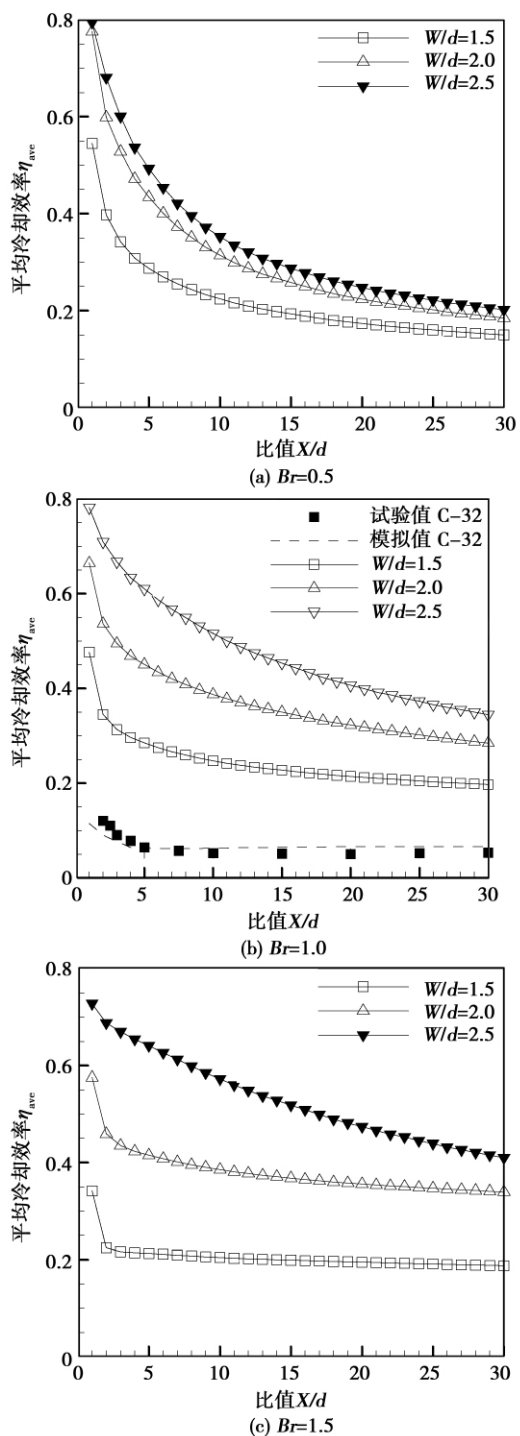


图 3 扇形孔出口宽度对径向平均冷却效率的影响
Fig. 3 Influence of the outlet width of a fan-shaped hole on the radially averaged cooling efficiency

2.2 面冷却效率

图 5 给出了吹风比 1.0 时,不同出口宽度射流下的冷却效率云图,气膜孔出口中心下游区是高冷

却效率区 随着气膜孔出口宽度的增加,高冷却效率区的宽度增加。 W/d 为 1.5 时,相邻两个孔之间存在明显的冷却薄弱区, W/d 为 2.0 时,冷却薄弱区的冷却效率明显得到了改善, W/d 为 2.5 时,基本实现了气膜全覆盖。在气膜孔下游方向,随着气膜孔出口宽度的增加,高冷却效率区越大,在 $W/d = 1.5$ 时 $x/d = 5$ 位置的冷却效率为 0.56,而在 $W/d = 2.5$ 时在 $x/d = 10$ 位置的冷却效率为 0.56。这说明,增加气膜孔出口宽度可以有效提高冷却效率。

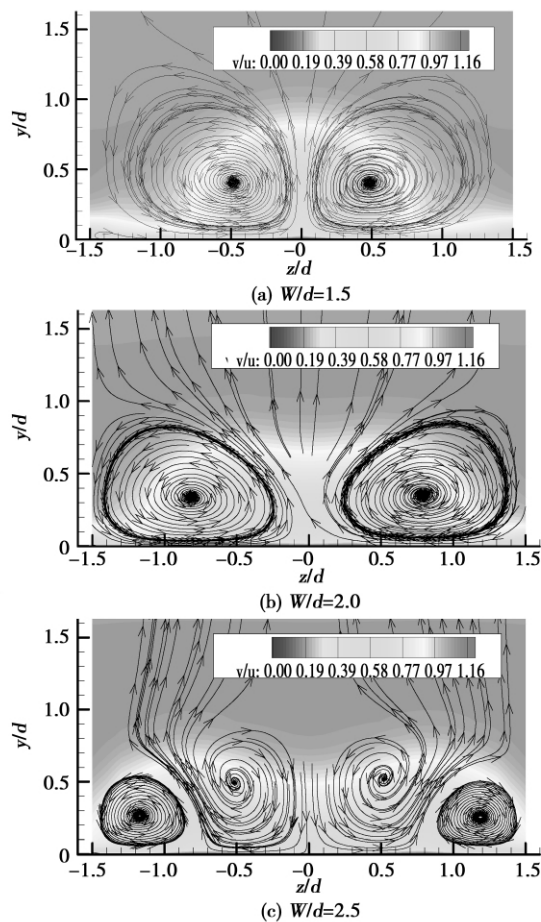


图 4 气膜孔下游 $x/d = 5$ 处速度矢量和无量纲速度

Fig. 4 Velocity vector and non-dimensional velocity at the downstream of an air film cooling hole at which $x/d = 5$

2.3 气膜冷却涡量

图 6 给出了吹风比 1.0 时气膜孔下游不同 x/d 截面位置 x 轴方向的最大涡量分布。由于冷气从扇形孔喷出后在 $y-z$ 平面形成的涡流是对称的, x 轴方向涡量平均值应该为 0,所以本文提取了 $y-z$ 平面涡量最大值,气膜孔出口宽度增加使涡量减小,在

不同的气膜孔出口宽度下, 涡量的最大值基本都发生在 $x/d = 2$ 的位置, 而后随着 x/d 增加, 涡量迅速减小. 在 $x/d = 15$ 位置, 不同出口宽度下的涡量基本相同. 这意味着在 $x/d > 15$ 的区域, 冷气和燃气的掺混非常弱, 冷气被燃气加热速度降低, 冷却效率沿着流动方向降低的速度减慢.

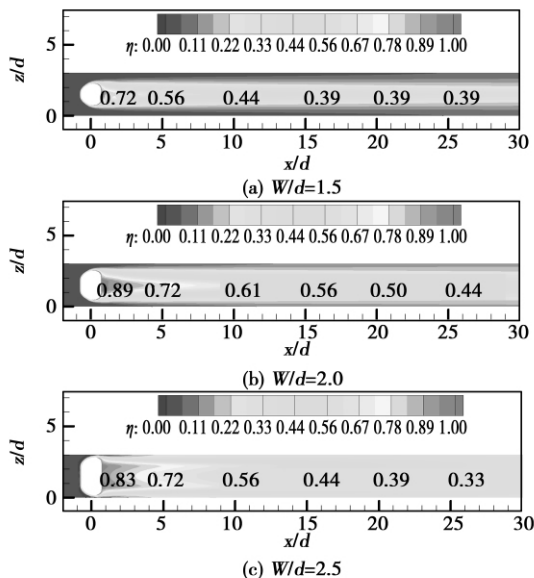


图 5 吹风比 1.0 时的冷却效率云图
 Fig. 5 Atlas of the cooling efficiencies when the air blowing ratio is 1.0

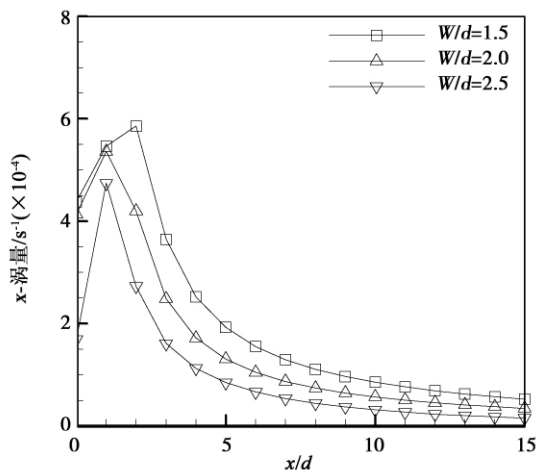


图 6 气膜孔下游的最大涡量分布
 Fig. 6 Distribution of the maximal vorticity at the downstream of an air film hole

3 结 论

采用数值模拟方法, 研究了不同出口宽度扇形孔射流冷却效率, 获得结论为:

(1) 出口宽度增加不但减小了涡量, 还改变了涡流方向. W/d 为 1.5 和 2.0 时, 气膜孔中心下游冷气流线由壁面指向燃气, 导致冷气卷吸到冷气底层. W/d 为 2.5 时, 气膜孔中心下游冷气流线由燃气指向壁面, 将燃气推到孔两侧, 减少了与冷气掺混的燃气量.

(2) 气膜孔出口宽度越大, 冷却效率越大. 随着吹风比的增加, 气膜孔出口宽度对冷却效率影响越明显.

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(丛 敏 编辑)

spectively and the above-mentioned blades exhibit a good adaptability to various incidence angles. **Key words:** turbine cascade ,rotor cascade ,rear-loaded blade profile ,incidence angle

超音速喷嘴变工况性能分析与试验研究 = **Analysis and Experimental Study of the Off-design Operating Condition Performance of a Supersonic Nozzle** [刊, 汉] GAO Yi-qiu ,LI Yi-xing (CSIC No. 704 Research Institute ,Shanghai ,China ,Post Code: 200031) ,WANG Hui (China Gas Turbine Research Institute ,Aviation Industry Corporation of China ,Youjiang ,China ,Post Code: 621703) // Journal of Engineering for Thermal Energy & Power. - 2012 27(4) . -411 ~415

Usually ,supersonic nozzles are used in the Curtis stage of a marine steam turbine. Through a numerical simulation method ,the authors have performed an analysis of the off-design operating condition performance of a nozzle with the law governing a change of its velocity coefficients at various pressure ratios and its internal flow field being studied and the velocity coefficient(0.96) and total pressure recovery coefficient(0.87) of the nozzle under the design operating condition being obtained. To further know well the aerodynamic performance of the nozzle under discussion , a wind tunnel test of its plane cascade was conducted. The distribution of the Mach numbers on the surface of the blade at different pressure ratios was measured and the characteristics of the flow field inside the passage of the cascade were analyzed in combination with the numerical simulation results. In the cross section at the outlet of the supersonic nozzle ,a turning point of the Mach number will be produced on the suction surface due to the influence of the expansion wave group and other factors and will gradually shift backward with the pressure ratio. When the pressure ratio deviates from its design one ,a steam flow separation will emerge on the suction surface of the cascade. Through a schlieren photo of the S_1 stream plane of the cascade ,the composition of the wave system in the flow passage of the cascade were observed and exhibited a relatively high agreement with the numerical simulation results. The test results verifies that the numerical calculation results are correct and make designers more visually learn the flow characteristics inside the supersonic nozzle. **Key words:** marine steam turbine ,supersonic ,nozzle ,numerical simulation ,wind tunnel test ,velocity coefficient ,Mach number

扇形孔出口宽度对气膜冷却效率影响 = **Influence of the Outlet Width of the Fan-shaped Holes on the Air Film Cooling Efficiency** [刊, 汉] ZHANG Wei ,LI Guang-chao ,WU Chao-lin (College of Power and Energy Source Engineering ,Shenyang University of Aeronautics and Astronautics ,Shenyang ,China ,Post Code: 110136) ,DENG Ming-chun(Internal Flow and Heat Transfer Research Department ,Shenyang Engine Design Research Institute ,She-

nyang ,China ,Post Code: 110085) //Journal of Engineering for Thermal Energy & Power. - 2012 27(4) . - 416 ~ 419

Numerically simulated and studied were the air film cooling efficiency of the jet flow from fan-shaped holes with various widths at the outlet at different air blowing ratios. The emphasis was placed on an analysis of the influence of the vortex configuration and vorticity on the cooling efficiency. It has been found that to increase the outlet width will not only decrease the vorticity but also change the direction of the vortex. When the outlet width increases to 2.5 times bigger than that of the inlet diameter ,the streamlines of the cooling air at the downstream of the hole center will point to the wall surface from the fuel gas ,effectively improving the attachment property of the cooling air. The bigger the outlet width of the air film hole ,the higher the cooling efficiency. With an increase of the air blowing ratio ,the influence of the outlet width of the air film hole on the cooling efficiency will become more conspicuous. **Key words:** turbine blade ,air film cooling ,vorticity ,cooling efficiency ,numerical simulation

汽轮机热力试验中参数测量误差对性能指标影响的算法研究 = **Study of the Algorithm for Calculating the Influence of the Measurement Errors of the Parameters During a Thermal Test of a Steam Turbine on Its Performance Indexes** [刊 ,汉] XIE Lin-gui ,CHENG Mao-hua ,GAO Wei (National Engineering Research Center for Unit Vibration in Thermal Power Plants ,Southeast University ,Nanjing ,China ,Post Code: 210096) // Journal of Engineering for Thermal Energy & Power. - 2012 27(4) . - 420 ~ 423

Based on the error transfer theory and by using the numerical analytic theory such as Taylor series and numerical differential etc. ,presented was a numerical calculation method for evaluating the influence of the measurement errors of parameters during the thermodynamic performance test of a steam turbine on its performance indexes not needing any parameter correction curves and performance index calculation function partial differential equations. With the test data of a 600 MW unit in a power plant serving as the object ,the algorithm in question was used to calculate the influence of the measurement errors of parameters on the performance indexes and the calculated results were compared with those obtained by using the conventional calculation method. It has been found that the uncertainty error of the heat rate of the unit calculated by using the numerical error analytic method proposed by the authors and the conventional calculation method is only 0.023% ,which is very small and such a result is reliable ,indicating that the error analytic calculation method under discussion is correct. Therefore ,it is an effective method for calculating the influence of measurement errors of parameters on the performance index of a unit when the conditions for using the conventional error analytic method are difficult to meet. **Key words:** steam turbine ,thermodynamic per-